

On the accuracy of the rotation form in simulations of the Navier-Stokes equations

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Outline

- Joint work with W. Layton, C. Manica, M. Neda and M. Olshanskii
- NSE and its nonlinearity
- Discrete nonlinearity forms: convective, skew-symmetric, rotational
- Examples that show drastic differences; rotational form bad!
- Discussion of possible cause
- Fix proposed: Grad-div stabilization
- Examples that show fix works



Navier-Stokes equations

- NSE in convective form

$$u_t + u \cdot \nabla u + \nabla p - Re^{-1} \Delta u = f, \quad \nabla \cdot u = 0$$

- Vector identity:

$$u \cdot \nabla u = -u \times (\nabla \times u) + \frac{1}{2} \nabla |u|^2$$

- Using Bernoulli pressure $\tilde{p} = p + \frac{1}{2} |u|^2$, NSE in rotational form given by

$$u_t - u \times (\nabla \times u) + \nabla \tilde{p} - Re^{-1} \Delta u = f, \quad \nabla \cdot u = 0$$



FEM scheme for NSE

- Naive Crank-Nicolson FEM scheme for NSE w/ convective form of NSE: $\forall (v_h, q_h) \in (X_h, Q_h)$

$$\begin{aligned} & \frac{1}{\Delta t} (u_h^{n+1} - u_h^n, v_h) + (u_h^{n+1/2} \cdot \nabla u_h^{n+1/2}, v_h) \\ & - (p_h^{n+1/2}, \nabla \cdot v_h) + Re^{-1} (\nabla u_h^{n+1/2}, \nabla v_h) = (f(t^{n+1/2}), v_h) \\ & (\nabla \cdot u_h^{n+1}, q_h) = 0 \end{aligned}$$

- This scheme known to be unstable



Variations of NSE scheme

- For stability, scheme often altered
- Skew symmetric form:

$$(u_h \cdot \nabla u_h, v_h) \rightarrow \frac{1}{2}(u_h \cdot \nabla u_h, v_h) - \frac{1}{2}(u_h \cdot \nabla v_h, u_h)$$

- Rotational form:

$$(u_h \cdot \nabla u_h, v_h) - (p_h, \nabla \cdot v_h) \rightarrow \\ - (u_h \times (\nabla \times u_h), v_h) - (\tilde{p}_h, \nabla \cdot v_h)$$

- In continuous case all 3 forms are equivalent



Variations of NSE scheme

- For both skew symmetric and rotational form schemes, we can prove

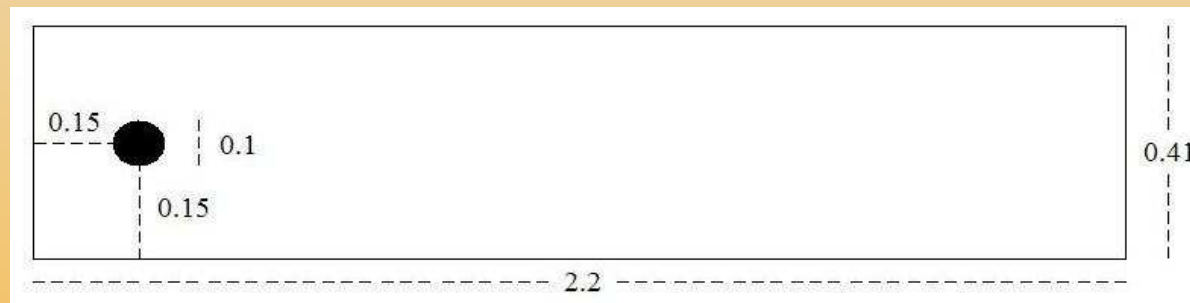
$$\|u_h^M\|^2 + \nu \Delta t \sum_{n=0}^{M-1} \|\nabla u_h^{n+1/2}\|^2 \leq C(Re, f, T, u_0)$$

- So both schemes are stable and conserve energy
- There are situations where one may want to use the rotational form scheme instead of convective form
 - Conservation of helicity
 - More robust preconditioners for large Reynolds numbers available
 - However, Rotational form can give bad results



Skew-symmetric vs. Rotational

Channel Flow around a cylinder benchmark problem
Domain:



The time dependent inflow and outflow profile are

$$u_1(0, y, t) = u_1(2.2, y, t) = \frac{6}{0.41^2} \sin(\pi t/8) y(0.41 - y)$$

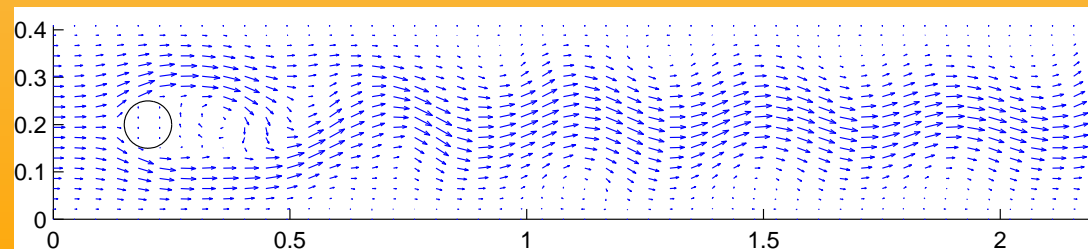
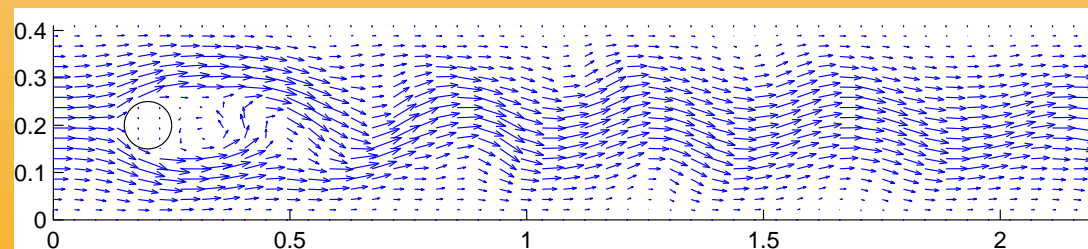
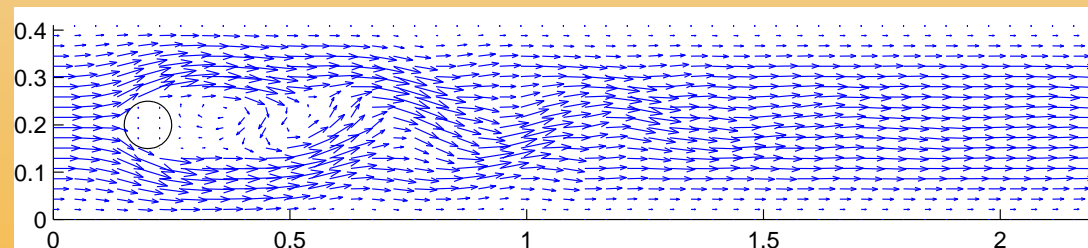
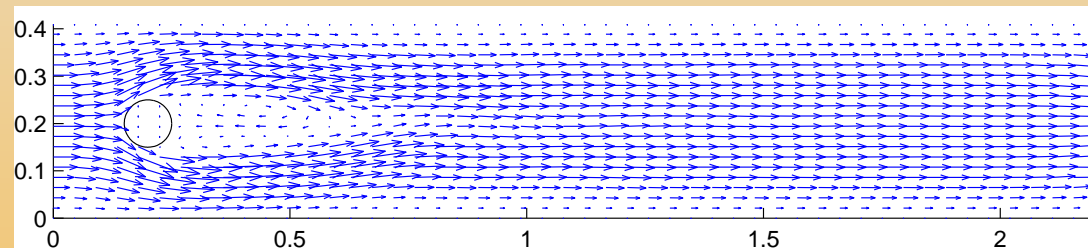
$$u_2(0, y, t) = u_2(2.2, y, t) = 0.$$

No slip boundaries, 0 initial condition, $0 \leq Re(t) \leq 100$.



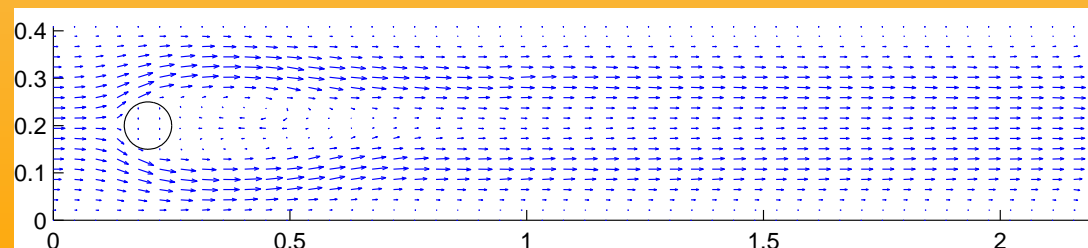
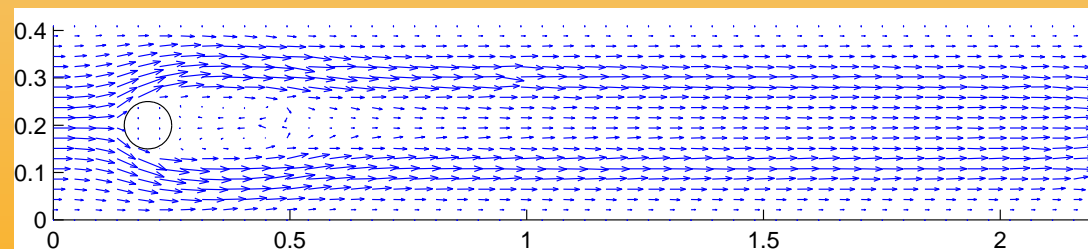
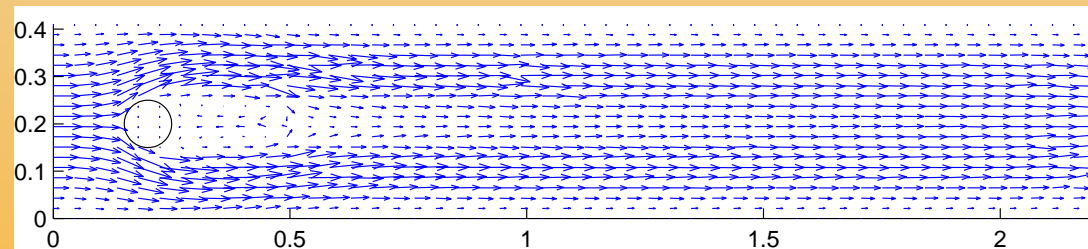
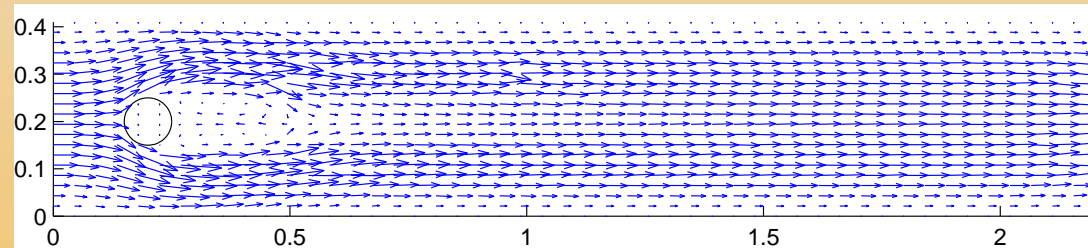
Skew-symmetric vs. Rotational

From (P_2, P_1) skew-symm scheme at $t=4,5,6,7$ (Good!)



Skew-symmetric vs. Rotational

(P_2, P_1) rotational form, same mesh, $t=4,5,6,7$ (Bad!)



Skew-symmetric vs. Rotational

- How can we get such different answers?
 - Increased pressure error for rotational form
 - $\tilde{p} = p + \frac{1}{2} |u|^2$ more complex than p
 - Boundary layers in Bernoulli pressure
 - On meshes where p_h is resolved, \tilde{p}_h may not be
 - For (P_k, P_{k-1}) elements, “some of u ” gets approximated with P_{k-1} for Bernoulli pressure
- Roughly speaking, if this pressure term is dominant, velocity error scales with $\text{Re} * \text{pressure error}$
- Similar problem in Stokes equations can be fixed with grad-div stabilization (Olshanskii/Reusken)
- Analysis of pressure term same in Stokes and NSE...



Fixing Rotational Form

- Consider Stokes problem, where there is same pressure term on RHS of velocity error equation

$$\begin{aligned} -Re^{-1}\Delta u + \nabla p &= f, \quad \nabla \cdot u = 0 \text{ in } \Omega \\ u &= 0 \text{ on } \partial\Omega \end{aligned}$$

- FEM scheme: $\forall v_h \in X_h, q_h \in Q_h$, solve

$$\begin{aligned} Re^{-1}(\nabla u_h, \nabla v_h) - (p_h, \nabla \cdot v_h) + (q_h, \nabla \cdot u_h) &= (f, v_h) \\ (\nabla \cdot u_h, q_h) &= 0 \end{aligned}$$



Fixing Rotational Form

- Typical velocity error bound for scheme:

$$\begin{aligned}\|\nabla(u - u_h)\| &\leq C\left(\inf_{v \in X_h} \|\nabla(u - v_h)\| + Re \inf_{q_h \in Q_h} \|p_h - p\|\right) \\ &\leq Ch^2(\|\nabla\nabla u\| + Re * \|\nabla p\|)\end{aligned}$$

- Olshanskii / Reusken: Add grad-div stabilization, $\gamma(\nabla \cdot u_h, \nabla \cdot v_h)$ to LHS of FEM scheme for Stokes equations, with $\gamma = O(1)$.
- New error equation:

$$\|\nabla(u - u_h)\| \leq Ch^2\sqrt{Re}(\|\nabla\nabla u\| + \|\nabla p\|)$$



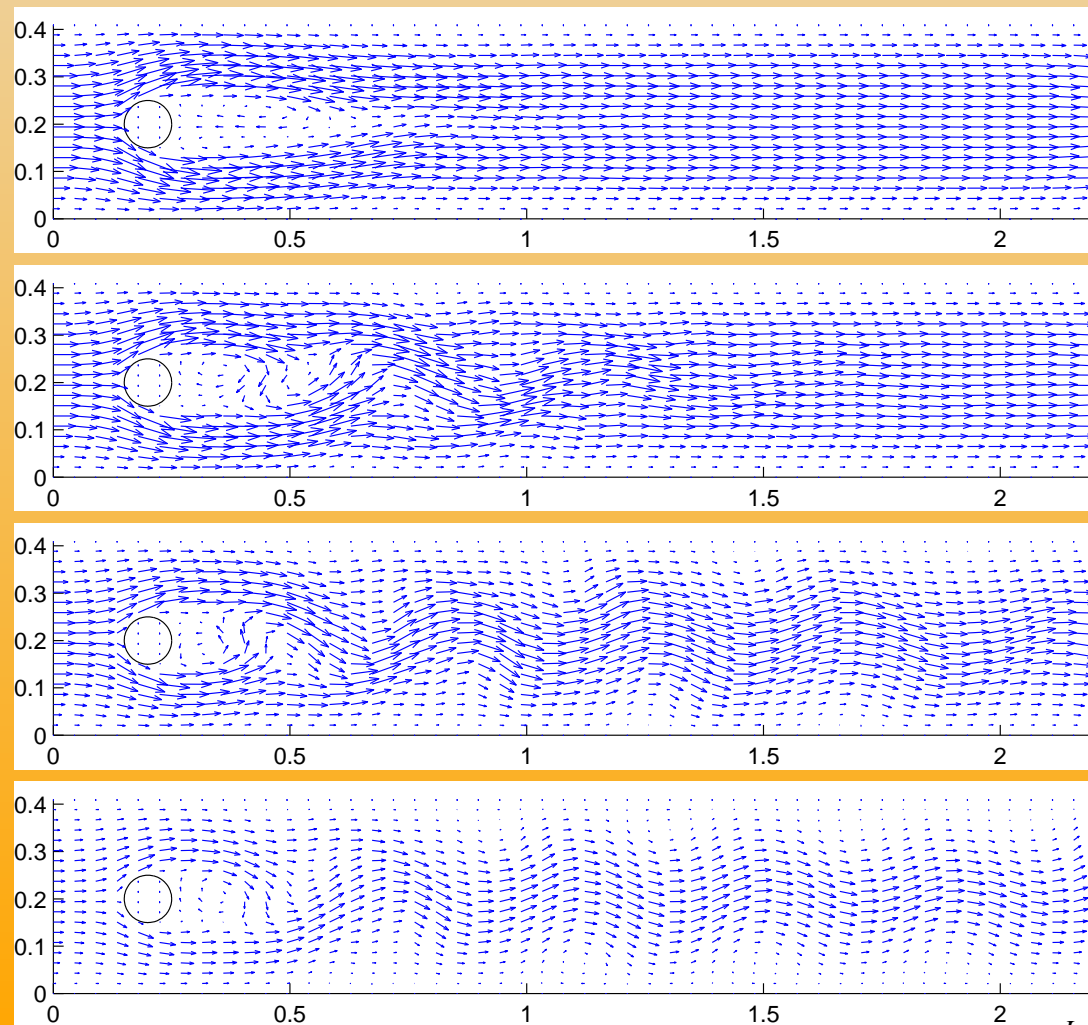
Fixing Rotational Form

- In short, the grad div term lets you handle $\inf_{q_h \in Q_h} \|p - q_h\| \|\nabla \cdot (u_h - U)\|$ in a different way
- Without stabilization, you have to “hide” the $\|\nabla \cdot (u_h - U)\|$ part on the left hand side in the $Re^{-1} \|\nabla(u_h - U)\|^2$ term
 - Creates a $Re * \inf_{q_h \in Q_h} \|p - q_h\|^2$ on the RHS
- With stabilization, will have a $\|\nabla \cdot (u_h - U)\|^2$ on the LHS, and so only a $\inf_{q_h \in Q_h} \|p - q_h\|^2$ will be left on the RHS
- Dependence of velocity error on pressure error is greatly reduced.



Fixing Rotational Form

From (P_2, P_1) grad-div stabilized rotational form at $t=4,5,6,7$ (Good!)



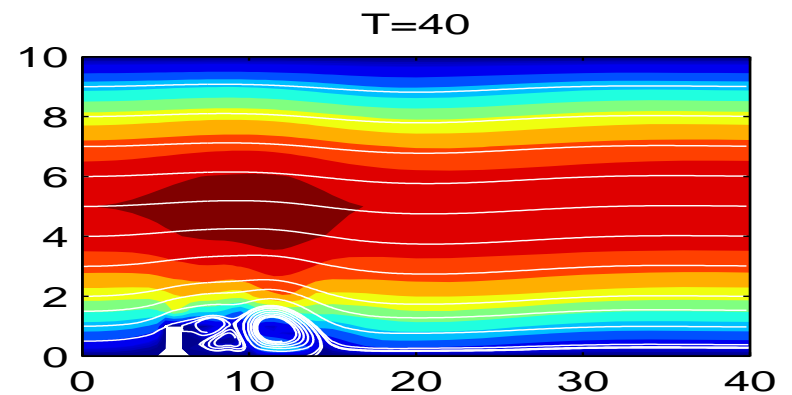
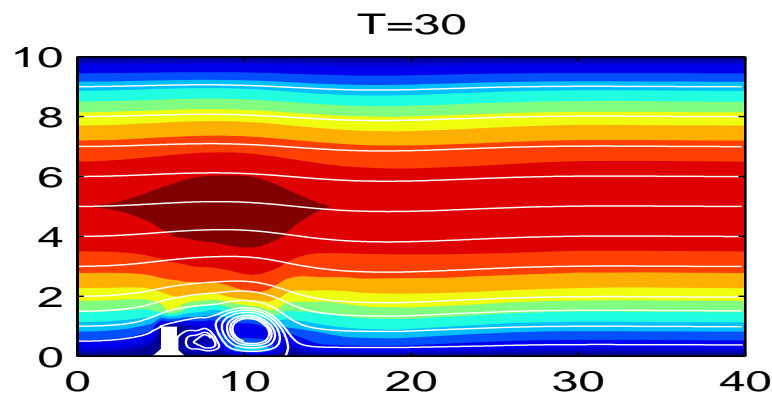
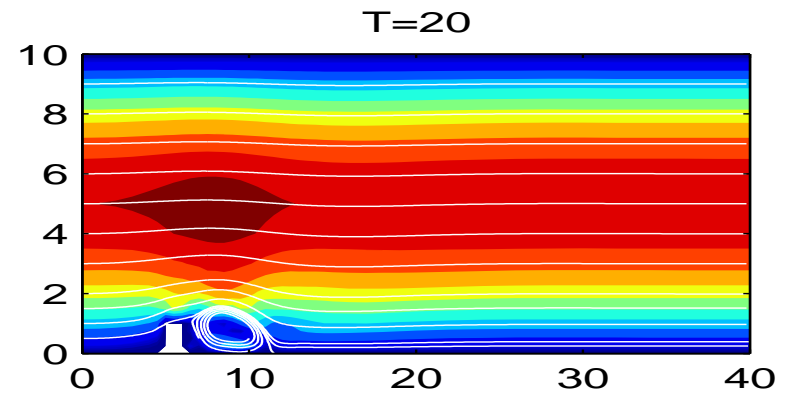
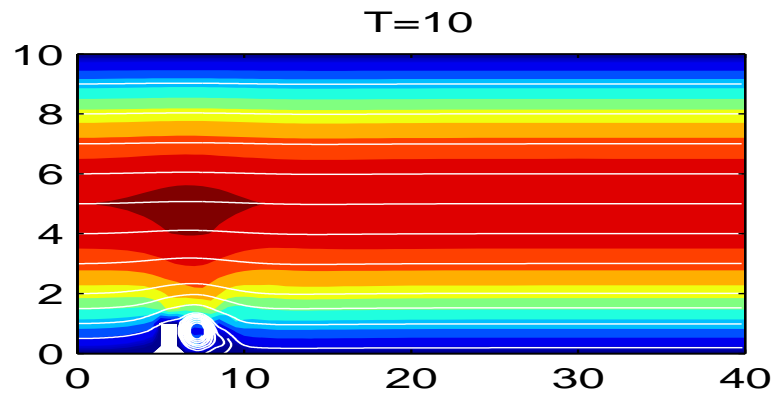
Flow over a Step

- Second test problem: Benchmark channel flow over a step
- FreeFem, (P_2, P_1) elements
- $Re=600$
- Dirichlet parabolic inflow, do-nothing outflow
- No slip sides
- Eddies should form behind step, detach, and new ones should form



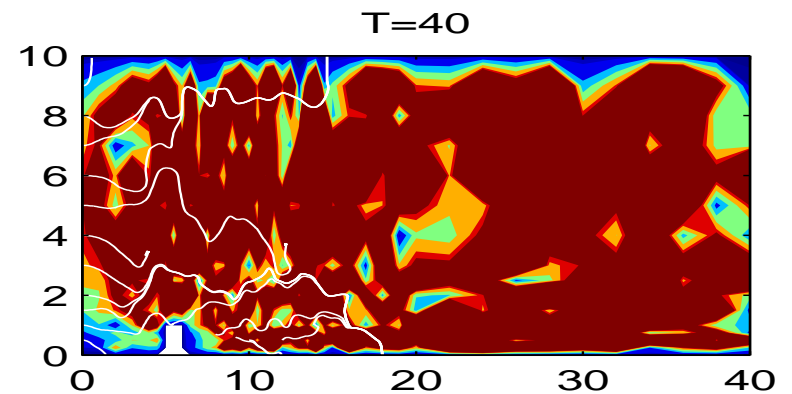
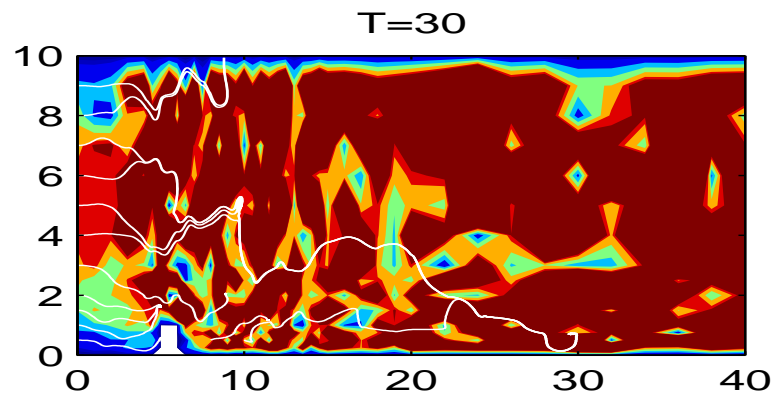
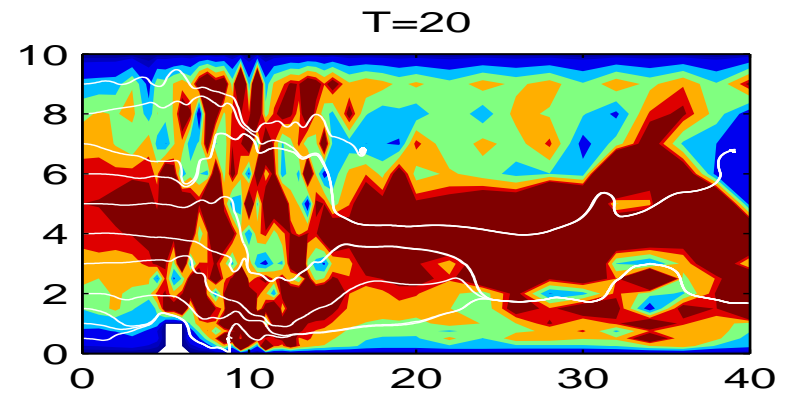
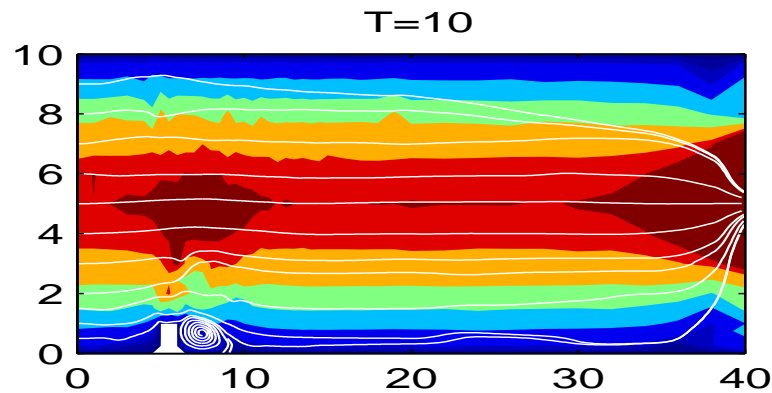
Flow over a Step

NSE with skew-symmetric form of nonlinearity



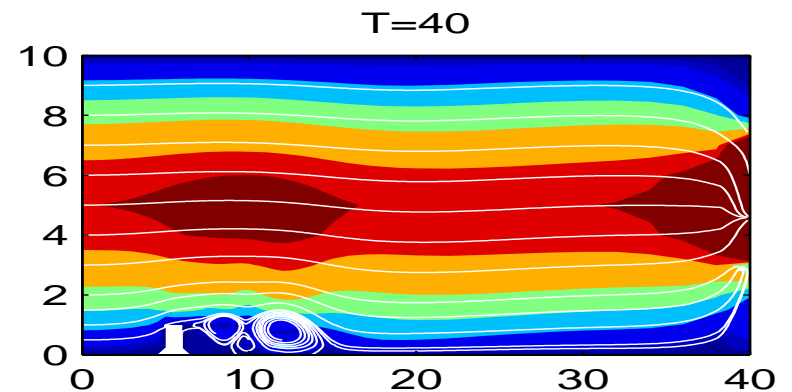
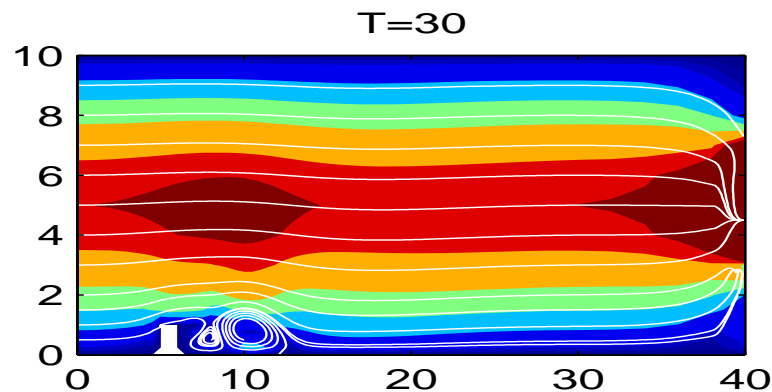
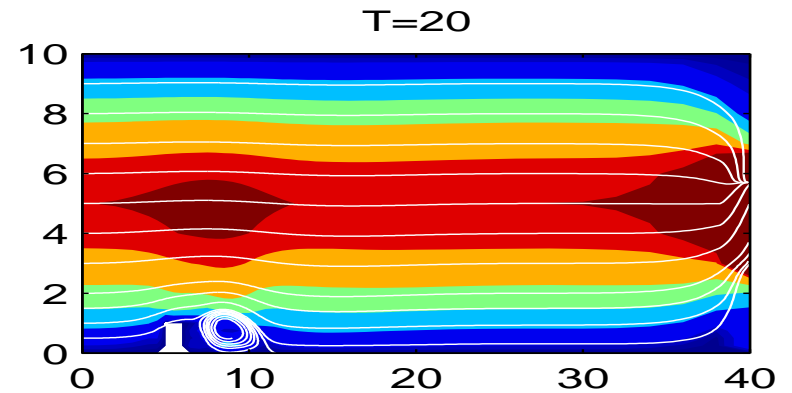
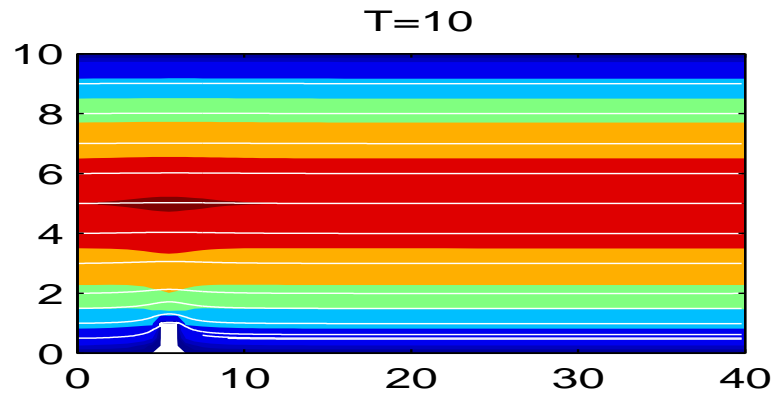
Flow over a Step

NSE w/ rotational form of nonlinearity



Flow over a Step

NSE w/ rotational form and grad-div stabilization;
unresolved issue w/ what is correct outflow bdry
condition



Conclusions

- There are cases where skew-symmetric FEM scheme tends to be much more accurate than rotational form scheme
- Difference is due to increased error in Bernoulli pressure, which in turn increases velocity error
- Adding grad-div stabilization can lessen the effect, especially for higher Re , thus reducing error in velocity field
- We are NOT claiming rotation + grad-div is best, but if you want to use rotational form, adding grad-div can greatly improve accuracy.

